

August 20, 2014 0111910030

Mr. Robert Hull Senior AQ Engineer Bay Area Air Quality Management District 939 Ellis Street San Francisco, CA 94109

### **SUBJECT: RESPONSE TO QUESTIONS ABOUT LEHIGH CUPERTINO HRA**

Dear Ted:

The purpose of this letter is to respond to the two comment letters from Mr. Frank R. Freedman, as requested by the Bay Area AQMD. These include the first letter ("Letter A") dated June 2, 2014 and the second letter ("Letter B") dated July 16, 2014. Each of the items will be discussed in turn.

In Letter B, there is a request to provide a protocol for HRA changes made as a function of the comments received. AQMD has also made this request. Thus, in this letter response, the protocol (explanation of methods used) is incorporated in each response item below.

### **Item #1—Wind gust speed for wind erosion calculations (Item #1 in Letter B, per Item A3 in Letter A, dated June 2, 2014—Note that the first two items in Letter A are being addressed by BAAQMD.)**

It has been confirmed that the wind gust speed calculations used in wind erosion emission estimates incorporated in the December 2013 HRA used an adjustment factor corresponding to a wind monitor elevation that was not consistent with the measurement height for the dataset used. That dataset was obtained from an offsite monitoring network whose data was posted on a website. In order to correct this wind monitor elevation adjustment factor, the meteorological monitoring data is required, and, this original data is no longer available. However, onsite wind data, a preferred source, is available. The use of this onsite data was proposed to BAAQMD and, hence, per instructions received from BAAQMD, the on-site wind gust data for July 1, 2010 through June 30, 2011 has now been used for wind gust speed calculations.



Mr. Robert Hull Bay Area AQMD August 20, 2014 Page 2

### **Item #5—Operating hours for quarry roads and other roads (Item #5 in Letter B, dated July 16, 2014)**

As of 2009, the quarry only operated five days per week, and this operating schedule was taken into account in wind erosion calculations. It is now assumed that all plant sections, including the quarry, operate 7 days/week, and thus the wind erosion calculations have been revised accordingly.

Both of these changes (wind gust correction and operating hours correction) have been incorporated in the wind erosion calculations, and the resulting change in emission factors is shown in the following table. These changes have been incorporated in the updated HRA emission rate calculations.



Note,

1. Stockpile emissions a function of stockpile geometry. Example shown above is for the bauxite stockpile.

### **Item #2—Wind speed for material handling from stockpiles emission calculations (Item #2 in Letter B, dated July 16, 2014):**

The same rationale for use of the onsite wind data set is applicable for this Item as for Item #1 above. This change has been incorporated in the updated HRA emission rate calculations.





Mr. Robert Hull Bay Area AQMD August 20, 2014 Page 3

### **Item #3—Benzene emission factor recalculation (Item #3 in Letter B):**

The results from kiln benzene source tests in 2009, 2010, and 2012, which were used in the December 2013 HRA, were re-evaluated for use in developing a benzene emission factor with the units "pounds of benzene emitted per short ton of clinker produced". These source test results contained all of the data necessary to prepare such a factor. The results were checked for data abnormalities, and then used to calculate this factor. The results of this recalculation indicate a benzene emission rate which is higher than that used in the December 2013 HRA. The detailed results are contained in an attachment to this letter. This change has been incorporated in the updated HRA emission rate calculations.

### **Item #4—Hexavalent chromium emissions from feed materials and stockpiles (Item #4 in Letter B):**

Under the AB2588 program, in cases where lab test results show non-detect values for a certain compound (in this case, hexavalent chromium) in a specified material, and engineering judgment and experience is that the compound will not be present in the material because of its production and handling, it is acceptable to use a zero value for the compound for that material. Because hexavalent chromium is generally only formed from trivalent chromium during combustion at high temperatures or under other severe oxidizing conditions, it is unlikely that hexavalent chromium will be present in the following raw materials because they have not previously been exposed to those conditions: Quarry overburden, limestone (all grades), iron ore, coal, coke, gypsum, and slag. Thus, no changes have been made to the updated HRA emission calculations for these materials.

### **Item #6—Diesel PM emission factor for emergency generator (Item #6 in Letter B):**

It has been confirmed that the diesel emission factor used in the December 2013 HRA came from AP-42 Table 3.4-2 rather than 3.4-1. This is because the AP-42 section instructs the reader to use Table 3.4-2 for PM emission factor. Therefore, the emission factor selected was appropriate and acceptable to use. Thus, no changes have been made to the updated HRA emission calculations for these sources.

### **Item #7—Diesel emissions from mobile sources (Item #7 in Letter B)—To be addressed by BAAQMD**



Mr. Robert Hull Bay Area AQMD August 20, 2014 Page 4

**Item #8—Request for protocol (Item #8 in Letter B)—** The above explanations have incorporated detailed descriptions of the methods used to evaluate topics of interest, and where required, any changes made to values used in the updated HRA emission calculations.

If you have questions regarding this matter, please call Anne McQueen with AMEC Environment & Infrastructure at (949) 574-7082.

Very truly yours,

Caughablelly

 $\alpha \mathcal{M}_l$  due  $\gamma$ <br>Caryn A. Kelly Anne McQueen, Ph.D., P.E.

Senior Toxicologist Senior Associate Engineer

Enclosures:

- 1. 2010 Lehigh On-Site Met Data (July 1, 2010 through June 30, 2011)
- 2. Wind Erosion Emission Factor Calculation for Unpaved Road and Bauxite Stockpile Using 2010 Lehigh On-Site Met Data
- 3. Material Handling Emission Factor Calculations
- 4. Stationary Emergency Diesel Generator Emission Factor Support
- 5. Kiln Benzene emission factor calculation
- 6. Updated 2013 HRA emission rate tables for the kiln and fugitive sources

Met Data

 $8/15/2014$ 

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:\11191.000.0\CEIR.2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls[tab]

 $\frac{1}{3}$ 

 $\ddot{\phantom{0}}$ 

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:\11191.000.0\CEIR.2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls[tab]

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:\11191.000.0\CEIR.2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls[tab]

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:11191.000.0\CEIR.2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls[tab]

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA

 $\frac{1}{2}$ 



K:\11191.000.0\CEIR 2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls(tab)

Page 5 of 8

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:11191.000.0\CEIR 2008\Emission calcs\Hanson Appendix B Tables rev onsite.xis(lab)

Page 6 of 8

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



K:11191.000.0ICEIR 2008IEmission calcsIHanson Appendix B Tables rev onsite.xis(tab)

# COMPARISON OF WIND DATA USED IN 2008 CEIR AND LEHIGH ON-SITE DATA<br>SUNNYVALE, CA



Notes.<br>1. Wind data used in original CEIR downloaded from www.wunderground.com, but specific location not specified.<br>2. On-site data used. Per BAAQMD, use 7/1/2010 to 6/30/2011.

Page 8 of 8

K:\11191.000.0\CEIR 2008\Emission calcs\Hanson Appendix B Tables rev onsite.xis[tab]

Wind Erosion

 $8/12/10/4$ 

AMEC Geomatrix, Inc.<br>Page 1 of 11

TABLE C-1

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility

 $0.5$  $k(PM10) =$ threshold friction velocity  $(u^*)_ =$ 

<= weekdays only  $\epsilon$ =365 days  $\leq$  AP-42<sup>2</sup>  $\frac{0.62}{0.73}$ ton/acre\*yr<br> $\frac{0.73}{1.22}$  ton/acre\*yr  $Ef(PM10)=$ <br> $Ef(PM10)=$ 



Onsite<br>De prototata

K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls\Roads Wind Rev Lehigh 07.14

नन

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



AMEC Geomatrix, Inc.<br>Page 2 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



AMEC Geomatrix, Inc.<br>Page 3 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cement Company<br>Cupertino Facility

 $\ddot{\phantom{0}}$ 



K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls\Roads Wind Rev Lehigh 07:14

AMEC Geomatrix, Inc.<br>Page 4 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cement Company<br>Cupertino Facility



AMEC Geomatrix, Inc.<br>Page 5 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xIs\Roads Wind Rev Lehigh 07.14

AMEC Geomatrix, Inc.<br>Page 6 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cement Company<br>Cupertino Facility



K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsile v3.xls\Roads Wind Rev Lehigh 07:14

AMEC Geomatrix, Inc.<br>Page 7 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



AMEC Geomatrix, Inc.<br>Page 8 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



AMEC Geomatrix, Inc.<br>Page 9 of 11

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cement Company<br>Cupertino Facility

 $\frac{1}{2} \left( \frac{1}{2} \right)^{2}$ 

 $\sim$ 



AMEC Geomatrix, Inc.<br>Page 10 of 11

 $\hat{\boldsymbol{\gamma}}$ .<br>Ma

### WIND EROSION FROM UNPAVED ROADS<br>Lehigh Southwest Cernent Company<br>Cupertino Facility



 $\sqrt{3}$ 

 $Ef(PM10)=$ 

Notes:

1. For u<sup>+</sup> used gust speed. Data was taken at a height of 10 m with on-site meteorlogical data.<br>2. u<sup>+</sup>, obtained from Table 13.2.5-2 AP-42

Wyer

3. Maximum wind gust speed was used in place of fastest mile.

internations only

AMEC Geomatrix, Inc.<br>Page 11 of 11

K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls\Roads Wind Rev Lehigh 07:14



k (PM10) =<br>threshold friction velocity (u\*<sub>i</sub>) =

 $\frac{0.5}{1.12}$  m/s

Page 1 of 10

K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls

Onsite 2010 Met data

k (PM10) =<br>threshold friction velocity (u\*<sub>i</sub>) =

 $\frac{0.5}{\sqrt{12}}$  m/s



k (PM10) =<br>threshold friction velocity (u<sup>\*</sup>i) =

**The Contract of the Contract o** 



 $0.5$ 

 $\overline{\phantom{a}}$ 



f

 $\frac{1}{1}$ 

k (PM10) =<br>threshold friction velocity (u\*<sub>i</sub>) =



S

 $0.5$ 





 $0.5$ 

k (PM10) =<br>threshold friction velocity (u\*;) =





 $\frac{1}{2}$ 



k (PM10) =<br>threshold friction velocity (u\*,) =

 $\frac{0.5}{1.12}$  m/s





k (PM10) =<br>threshold friction velocity (u\*,) =



 $0.5$ 



k (PM10) =<br>threshold friction velocity (u\*<sub>i</sub>) =

L

 $\overline{\phantom{a}}$ 

 $\frac{0.5}{1.12}$  m/s



K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls

k (PM10) =<br>threshold friction velocity (u<sup>+</sup><sub>t</sub>) =



 $0.5$ 





ign Soutnwest Cement Company<br>Cupertino Facility

Pile Subarea

k (PM10) =<br>threshold friction velocity (u\*,) =

 $\frac{0.5}{1.12}$  m/s



 $(g/m^2)$ 

 $(9/m^2)$  $\mathbf{a}^{\!-}$ 

ਠਾ ਠ ತ

> ठाठ ठांठ

5

 $\circ$ 

 $0.9$ ΙcΞ

 $0.6a$ 

 $0.2<sub>b</sub>$  $(g/m^2)$  $|\mathfrak{a}$ 

 $\frac{1}{2a}$  $\overline{(g/m^2)}$ ۵ĩ

 $\frac{0.9}{\text{p}}$ <br>(g/m<sup>2</sup>)

 $\left(\frac{q}{m^2}\right)$  $\frac{6a}{p}$ 

Disregarding Weekends









Sum:<br>Pi x A<br>Ef =<br>PM10 Ef:<br>E (ton/yr)

K:\11191.000.0\Emissions Rev 2013\Lehigh Appendix Tables Apr 26 2013\_vonsite v3.xls

Material Handling

# COMPARISON OF MATERIAL HANDLING EMISSION FACTORS USING<br>ON-SITE WIND DATA<br>Lehigh Southwest Cement Company<br>Cupertino Facility



K:\11191.000.0\CEIR 2008\Emission calcs\Hanson Appendix B Tables rev onsite.xls

AMEC Geomatrix, Inc.

# MATERIAL HANDLING OF STOCKPILES

TABLE B-5

Lehigh Southwest Cement Company Cupertino Facility

> Equations for TSP Emission Factor as per EPA AP-42 Example for Quarry Overburden





Notes:

Wind speed data for 2008 assumed to be representative of typical year<br>1. Equation is from US EPA AP-42 discussion on aggregate handling and storage piles  $(13.2.4)$  equation 1

2. Values for moisture content taken from Table 13.2.4-1,  $AP-42$ . Assumed that all materials have moisture content of crushed limestone

TABLE B-5

AMEC Geomatrix, Inc.

### MATERIAL HANDLING OF STOCKPILES Lehigh Southwest Cement Company Cupertino Facility

**Equations for TSP Emission Factor as per EPA AP-42<sup>1</sup>**<br>Example for Quarry Overburden





Notes:

Wind speed data for 2008 assumed to be representative of typical year

Figure 1.1 and 1.1 an

J fem 6 Dresel

### 3.4 Large Stationary Diesel And All Stationary Dual-fuel Engines

### 3.4.1 General

The primary domestic use of large stationary diesel engines (greater than 600 horsepower [hp]) is in oil and gas exploration and production. These engines, in groups of 3 to 5, supply mechanical power to operate drilling (rotary table), mud pumping, and hoisting equipment, and may also operate pumps or auxiliary power generators. Another frequent application of large stationary diesels is electricity generation for both base and standby service. Smaller uses include irrigation, hoisting, and nuclear power plant emergency cooling water pump operation.

Dual-fuel engines were developed to obtain compression ignition performance and the economy of natural gas, using a minimum of 5 to 6 percent diesel fuel to ignite the natural gas. Large dual-fuel engines have been used almost exclusively for prime electric power generation. This section includes all dual-fuel engines.

### 3.4.2 Process Description

All reciprocating internal combustion (IC) engines operate by the same basic process. A combustible mixture is first compressed in a small volume between the head of a piston and its surrounding cylinder. The mixture is then ignited, and the resulting high-pressure products of combustion push the piston through the cylinder. This movement is converted from linear to rotary motion by a crankshaft. The piston returns, pushing out exhaust gases, and the cycle is repeated.

There are 2 ignition methods used in stationary reciprocating IC engines, compression ignition (CI) and spark ignition (SI). In CI engines, combustion air is first compression heated in the cylinder, and diesel fuel oil is then injected into the hot air. Ignition is spontaneous because the air temperature is above the autoignition temperature of the fuel. SI engines initiate combustion by the spark of an electrical discharge. Usually the fuel is mixed with the air in a carburetor (for gasoline) or at the intake valve (for natural gas), but occasionally the fuel is injected into the compressed air in the cylinder. Although all diesel- fueled engines are compression ignited and all gasoline- and gas-fueled engines are spark ignited, gas can be used in a CI engine if a small amount of diesel fuel is injected into the compressed gas/air mixture to burn any mixture ratio of gas and diesel oil (hence the name dual fuel), from 6 to 100 percent diesel oil.

CI engines usually operate at a higher compression ratio (ratio of cylinder volume when the piston is at the bottom of its stroke to the volume when it is at the top) than SI engines because fuel is not present during compression; hence there is no danger of premature autoignition. Since engine thermal efficiency rises with increasing pressure ratio (and pressure ratio varies directly with compression ratio), CI engines are more efficient than SI engines. This increased efficiency is gained at the expense of poorer response to load changes and a heavier structure to withstand the higher  $pressures.<sup>1</sup>$ 

### 3.4.3 Emissions And Controls

Most of the pollutants from IC engines are emitted through the exhaust. However, some total organic compounds (TOC) escape from the crankcase as a result of blowby (gases that are vented from the oil pan after they have escaped from the cylinder past the piston rings) and from the fuel tank

and carburetor because of evaporation. Nearly all of the TOCs from diesel CI engines enter the atmosphere from the exhaust. Crankcase blowby is minor because TOCs are not present during compression of the charge. Evaporative losses are insignificant in diesel engines due to the low volatility of diesel fuels. In general, evaporative losses are also negligible in engines using gaseous fuels because these engines receive their fuel continuously from a pipe rather than via a fuel storage tank and fuel pump.

The primary pollutants from internal combustion engines are oxides of nitrogen  $(NO_x)$ , hydrocarbons and other organic compounds, carbon monoxide (CO), and particulates, which include both visible (smoke) and nonvisible emissions. Nitrogen oxide formation is directly related to high pressures and temperatures during the combustion process and to the nitrogen content, if any, of the fuel. The other pollutants, HC, CO, and smoke, are primarily the result of incomplete combustion. Ash and metallic additives in the fuel also contribute to the particulate content of the exhaust. Sulfur oxides also appear in the exhaust from IC engines. The sulfur compounds, mainly sulfur dioxide  $(SO<sub>2</sub>)$ , are directly related to the sulfur content of the fuel.<sup>2</sup>

### 3.4.3.1 Nitrogen Oxides -

Nitrogen oxide formation occurs by two fundamentally different mechanisms. The predominant mechanism with internal combustion engines is thermal  $NO_x$  which arises from the thermal dissociation and subsequent reaction of nitrogen  $(N_2)$  and oxygen  $(O_2)$  molecules in the combustion air. Most thermal  $NO<sub>x</sub>$  is formed in the high-temperature region of the flame from dissociated molecular nitrogen in the combustion air. Some  $\overline{NO}_{x}$ , called prompt  $NO_{x}$ , is formed in the early part of the flame from reaction of nitrogen intermediary species, and HC radicals in the flame. The second mechanism, fuel  $NO_x$ , stems from the evolution and reaction of fuel-bound nitrogen compounds with oxygen. Gasoline, and most distillate oils, have no chemically-bound fuel  $N_2$  and essentially all  $NO_x$  formed is thermal  $NO_x$ .

### 3.4.3.2 Total Organic Compounds -

The pollutants commonly classified as hydrocarbons are composed of a wide variety of organic compounds and are discharged into the atmosphere when some of the fuel remains unburned or is only partially burned during the combustion process. Most unburned hydrocarbon emissions result from fuel droplets that were transported or injected into the quench layer during combustion. This is the region immediately adjacent to the combustion chamber surfaces, where heat transfer outward through the cylinder walls causes the mixture temperatures to be too low to support combustion.

Partially burned hydrocarbons can occur because of poor air and fuel homogeneity due to incomplete mixing, before or during combustion; incorrect air/fuel ratios in the cylinder during combustion due to maladjustment of the engine fuel system; excessively large fuel droplets (diesel engines); and low cylinder temperature due to excessive cooling (quenching) through the walls or early cooling of the gases by expansion of the combustion volume caused by piston motion before combustion is completed.<sup>2</sup>

### 3.4.3.3 Carbon Monoxide -

Carbon monoxide is a colorless, odorless, relatively inert gas formed as an intermediate combustion product that appears in the exhaust when the reaction of CO to  $CO<sub>2</sub>$  cannot proceed to completion. This situation occurs if there is a lack of available oxygen near the hydrocarbon (fuel) molecule during combustion, if the gas temperature is too low, or if the residence time in the cylinder is too short. The oxidation rate of CO is limited by reaction kinetics and, as a consequence, can be accelerated only to a certain extent by improvements in air and fuel mixing during the combustion process. $2-3$ 

### 3.4.3.4 Smoke, Particulate Matter, and PM-10 -

White, blue, and black smoke may be emitted from IC engines. Liquid particulates appear as white smoke in the exhaust during an engine cold start, idling, or low load operation. These are formed in the quench layer adjacent to the cylinder walls, where the temperature is not high enough to ignite the fuel. Blue smoke is emitted when lubricating oil leaks, often past worn piston rings, into the combustion chamber and is partially burned. Proper maintenance is the most effective method of preventing blue smoke emissions from all types of IC engines. The primary constituent of black smoke is agglomerated carbon particles  $(soot)^2$ 

### $3.4.3.5$  Sulfur Oxides -

Sulfur oxide emissions are a function of only the sulfur content in the fuel rather than any combustion variables. In fact, during the combustion process, essentially all the sulfur in the fuel is oxidized to  $SO_2$ . The oxidation of  $SO_2$  gives sulfur trioxide  $(SO_3)$ , which reacts with water to give sulfuric acid  $(\hat{H_2}SO_4)$ , a contributor to acid precipitation. Sulfuric acid reacts with basic substances to give sulfates, which are fine particulates that contribute to PM-10 and visibility reduction. Sulfur oxide emissions also contribute to corrosion of the engine parts.  $2<sup>3</sup>$ 

Table 3.4-1 contains gaseous emission factors for the pollutants discussed above, expressed in units of pounds per horsepower-hour (lb/hp-hr), and pounds per million British thermal unit (lb/MMBtu). Table 3.4-2 shows the particulate and particle-sizing emission factors. Table 3.4-3 shows the speciated organic compound emission factors and Table 3.4-4 shows the emission factors for polycyclic aromatic hydrocarbons (PAH). These tables do not provide a complete speciated organic compound and PAH listing because they are based only on a single engine test; they are to be used only for rough order of magnitude comparisons.

Table 3.4-5 shows the  $NO_x$  reduction and fuel consumption penalties for diesel and dual-fueled engines based on some of the available control techniques. The emission reductions shown are those that have been demonstrated. The effectiveness of controls on a particular engine will depend on the specific design of each engine, and the effectiveness of each technique could vary considerably. Other  $NO<sub>x</sub>$  control techniques exist but are not included in Table 3.4-5. These techniques include internal/external exhaust gas recirculation, combustion chamber modification, manifold air cooling, and turbocharging.

### 3.4.4 Control Technologies

Control measures to date are primarily directed at limiting  $NO<sub>x</sub>$  and CO emissions since they are the primary pollutants from these engines. From a  $NO_x$  control viewpoint, the most important distinction between different engine models and types of reciprocating engines is whether they are rich-burn or lean-burn. Rich-burn engines have an air-to-fuel ratio operating range that is near stoichiometric or fuel-rich of stoichiometric and as a result the exhaust gas has little or no excess oxygen. A lean-burn engine has an air-to-fuel operating range that is fuel-lean of stoichiometric; therefore, the exhaust from these engines is characterized by medium to high levels of  $O_2$ . The most common  $NO_x$  control technique for diesel and dual fuel engines focuses on modifying the combustion process. However, selective catalytic reduction (SCR) and nonselective catalytic reduction (NSCR) which are post-combustion techniques are becoming available. Control for CO have been partly adapted from mobile sources.<sup>5</sup>

Combustion modifications include injection timing retard (ITR), preignition chamber combustion (PCC), air-to-fuel ratio, and derating. Injection of fuel into the cylinder of a CI engine initiates the combustion process. Retarding the timing of the diesel fuel injection causes the combustion process to occur later in the power stroke when the piston is in the downward motion and

combustion chamber volume is increasing. By increasing the volume, the combustion temperature and pressure are lowered, thereby lowering  $\overline{NO}_x$  formation. ITR reduces  $\overline{NO}_x$  from all diesel engines; however, the effectiveness is specific to each engine model. The amount of  $NO_x$  reduction with ITR diminishes with increasing levels of retard.<sup>5</sup>

Improved swirl patterns promote thorough air and fuel mixing and may include a precombustion chamber (PCC). A PCC is an antechamber that ignites a fuel-rich mixture that propagates to the main combustion chamber. The high exit velocity from the PCC results in improved mixing and complete combustion of the lean air/fuel mixture which lowers combustion temperature, thereby reducing  $NO_x$  emissions.<sup>5</sup>

The air-to-fuel ratio for each cylinder can be adjusted by controlling the amount of fuel that enters each cylinder. At air-to-fuel ratios less than stoichiometric (fuel-rich), combustion occurs under conditions of insufficient oxygen which causes  $NO<sub>x</sub>$  to decrease because of lower oxygen and lower temperatures. Derating involves restricting engine operation to lower than normal levels of power production for the given application. Derating reduces cylinder pressures and temperatures thereby lowering  $NO_x$  formation rates.<sup>5</sup>

SCR is an add-on  $NO_x$  control placed in the exhaust stream following the engine and involves injecting ammonia (NH<sub>3</sub>) into the flue gas. The NH<sub>3</sub> reacts with the NO<sub>x</sub> in the presence of a catalyst to form water and nitrogen. The effectiveness of SCR depends on fuel quality and engine duty cycle (load fluctuations). Contaminants in the fuel may poison or mask the catalyst surface causing a reduction or termination in catalyst activity. Load fluctuations can cause variations in exhaust temperature and  $NO<sub>x</sub>$  concentration which can create problems with the effectiveness of the SCR system.<sup>5</sup>

NSCR is often referred to as a three-way conversion catalyst system because the catalyst reactor simultaneously reduces  $NO_x$ , CO, and HC and involves placing a catalyst in the exhaust stream of the engine. The reaction requires that the  $O_2$  levels be kept low and that the engine be operated at fuel-rich air-to-fuel ratios.<sup>5</sup>

### 3.4.5 Updates Since the Fifth Edition

The Fifth Edition was released in January 1995. Revisions to this section since that date are summarized below. For further detail, consult the memoranda describing each supplement or the background report for this section.

Supplement A, February 1996

No changes.

Supplement B, October 1996

- The general text was updated.
- Controlled  $NO<sub>x</sub>$  factors and PM factors were added for diesel units.
- Math errors were corrected in factors for CO from diesel units and for uncontrolled  $NO<sub>x</sub>$  from dual fueled units.

Table 3.4-1. GASEOUS EMISSION FACTORS FOR LARGE STATIONARY DIESEL AND ALL STATIONARY DUAL-FUEL ENGINES<sup>a</sup>



from these levels. To convert from  $1b/hp-hr$  to kg/kw-hr, multiply by 0.608. To convert from  $1b/MMB$ tu to ng/l, multiply by 430. SCC = calculate only 1 of the 2 emission factors (e. g., enough information to calculate lb/MMBtu, but not lb/hp-hr). Factors are based on<br>averages across all manufacturers and duty cycles. The actual emissions from a particular  $\frac{1}{2}$  b Dual fuel assumes 95% natural gas and 5% diesel fuel.

<sup>e</sup> References 8-26. Controlled NO<sub>x</sub> is by ignition tinning retard.<br><sup>d</sup> Assumes that all sulfur in the fuel is converted to SO<sub>2</sub>. S<sub>1</sub> = % sulfur in fuel oil; S<sub>2</sub> = % sulfur in natural gas. For example, if sulfer Assumes 100% conversion of carbon in fuel to  $CO_2$  with 87 weight % carbon in diesel, 70 weight % carbon in natural gas, dual-fuel

mixture of 5% diesel with 95% natural gas, average BSFC of 7,000 Btu/hp-hr, diesel heating value of 19,300 Btu/lb, and natural gas Based on data from 1 engine, TOC is by weight 9% methane and 91% nonmethane.

où

Assumes that nonmethane organic compounds are 25% of TOC emissions from duat-fuel engines. Molecular weight of nonmethane gas

### Table 3.4-2. PARTICULATE AND PARTICLE-SIZING EMISSION FACTORS FOR LARGE UNCONTROLLED STATIONARY DIESEL ENGINES<sup>a</sup>



### **EMISSION FACTOR RATING: E**

Based on 1 uncontrolled diesel engine from Reference 6. Source Classification Code 2-02-004-01. The data for the particulate emissions were collected using Method 5, and the particle size distributions were collected using a Source Assessment Sampling System. To convert from lb/MMBtu to ng/J, multiply by 430. PM-10 = particulate matter  $\leq 10$  micrometers ( $\mu$ m) aerometric diameter.

b Particle size is expressed as aerodynamic diameter.

 $\frac{1}{2}$  Total PM-10 is the sum of filterable particulate less than 10  $\mu$ m aerodynamic diameter and condensable particulate.

d Total particulate is the sum of the total filterable particulate and condensable particulate.

 $\bf{a}$ 

### Table 3.4-3. SPECIATED ORGANIC COMPOUND EMISSION FACTORS FOR LARGE UNCONTROLLED STATIONARY DIESEL ENGINES<sup>a</sup>

### EMISSION FACTOR RATING: E



<sup>a</sup>Based on 1 uncontrolled diesel engine from Reference 7. Source Classification Code 2-02-004-01. Not enough information to calculate the output-specific emission factors of lb/hp-hr. To convert from lb/MMBtu to ng/J, multiply by 430. <sup>b</sup>Hazardous air pollutant listed in the Clean Air Act.

### Table 3.4-4. PAH EMISSION FACTORS FOR LARGE UNCONTROLLED STATIONARY DIESEL ENGINES<sup>a</sup>

### EMISSION FACTOR RATING: E



<sup>a</sup> Based on 1 uncontrolled diesel engine from Reference 7. Source Classification Code 2-02-004-01. Not enough information to calculate the output-specific emission factors of lb/hp-hr. To convert from lb/MMBtu to ng/J, multiply by 430.

b Hazardous air pollutant listed in the *Clean Air Act*.

 $\mathcal{A}$ 

 $\sim$ 

 $\sim$ 



### Table 3.4-5.  $\rm{NO_{X}}$  REDUCTION AND FUEL CONSUMPTION PENALTIES FOR LARGE STATIONARY DIESEL AND DUAL-FUEL ENGINES $^{\rm a}$

References 1,27-28. The reductions shown are typical and will vary depending on the engine and duty cycle.  $SCC = Source Classification Code.  $\triangle BSFC = change in brake-specific fuel$$ consumption.  $ND = no data$ .

### References For Section 3.4

- $\mathbb{L}$ H. I. Lips, et al., Environmental Assessment Of Combustion Modification Controls For Stationary Internal Combustion Engines, EPA-600/7-81-127, U. S. Environmental Protection Agency, Cincinnati, OH, July 1981.
- Standards Support And Environmental Impact Statement, Volume 1: Stationary Internal  $2.$ Combustion Engines, EPA-450/2-78-125a, U. S. Environmental Protection Agency, Research Triangle Park, NC, July 1979.
- M. Hoggan, et. al., Air Quality Trends in California's South Coast and Southeast Desert Air  $3.$ Basins, 1976-1990, "Air Quality Management Plan, Appendix II-B", South Coast Air Quality Management District, July 1991.
- Limiting Net Greenhouse Gas Emissions In the United States, Volume II: Energy Responses, 4. report for the Office of Environmental Analysis, Office of Policy, Planning and Analysis, Department of Energy (DDE), DOE/PE-0101 Volume II, September 1991.
- Snyder, R. B., Alternative Control Techniques Document- $NO_x$  Emissions from Stationary 5. Reciprocating Internal Combustion Engines, EPA-453/R-93-032, U. S. Environmental Protection Agency, Research Triangle Park, July 1993.
- C. Castaldini, Environmental Assessment Of NO<sub>x</sub> Control On A Compression Ignition Large 6. Bore Reciprocating Internal Combustion Engine, Volume I: Technical Results, EPA-600/7-86/001a, U. S. Environmental Protection Agency, Cincinnati, OH, April 1984.
- Pooled Source Emission Test Report: Oil And Gas Production Combustion Sources, Fresno  $7.$ And Ventura Counties. California, ENSR # 7230-007-700, Western States Petroleum Association, Bakersfield, CA, December 1990.
- Final Report For An Emission Compliance Test Program On Two Standby Generators Located 8. At American Car Company, Greenwich, CT, York Services Corp., 1987.
- Final Report For An Emission Compliance Test Program On A Standby Diesel Generator At 9. South Central Connecticut Regional Water Authority, West Haven, CT, York Services Corp., 1988.
- Air Emission From Stationary Diesel Engines For The Alaska Rural Electric Cooperative  $10.$ Association, Environmetrics, 1992.
- Compliance Test Report For Particulate Emissions From A Caterpillar Diesel Generator, St. 11. Mary's Hospital, Waterburg, CT, TRC Environmental Consultants, 1987.
- Compliance Measured Particulate Emissions From An Emergency Diesel Generator, Silorsky  $12.$ Aircraft, United Technologies, Stratford, CT, TRC Environmental Consultants, 1987.
- Compliance Test Report For Particulate Emissions From A Cummins Diesel Generator,  $13.$ Colonial Gold Limited Partnership, Hartford, CT, TRC Environmental Consultants, 1988.
- Compliance Test Report For Particulate Emissions From A Cummins Diesel Generator, 14. CIGNA Insurance Company, Bloomfield, CT, TRC Environmental Consultants, 1988.
- Compliance Test Report For Particulate Emission From A Waukesha Diesel Generator, Bristol  $15.$ Meyers, Wallinsford, CT, TRC Environmental Consultants, 1987.
- Compliance Test Report For Particulate Emissions From A Cummins Diesel Generator, 16. Connecticut General Life Insurance, Windsor, CT, TRC Environmental Consultants, 1987.
- 17. Compliance Measured Particulate Emissions From An Emergency Diesel Generator, Danbury Hospital, Danbury, CT, TRC Environmental Consultants, 1988.
- Compliance Test Report For Particulate Emissions From A Caterpillar Diesel Generator, 18. Colonial Metro Limited Partnership, Hartford, CT, TRC Environmental Consultants, 1988.
- 19. Compliance Test Report For Particulate Emissions From A Caterpillar Diesel Generator, Boehringer -Ingelheim Pharmaceuticals, Danbury, CT, TRC Environmental Consultants, 1988.
- Compliance Test Report For Emissions Of Particulate From An Emergency Diesel Generator,  $20.$ Meriden - Wallingford Hospital, Meriden, CT, TRC Environmental Consultants, 1987.
- Compliance Test Report Johnson Memorial Hospital Emergency Generator Exhaust Stack, 21. Stafford Springs, CT, ROJAC Environmental Services, 1987.
- Compliance Test Report Union Carbide Corporation Generator Exhaust Stack, Danbury, CT, 22. ROJAC Environmental Services, 1988.
- Compliance Test Report Hartford Insurance Company Emergency Generator Exhaust Stack, 23. Bloomfield, CT, ROJAC Environmental Services, 1987.
- Compliance Test Report Hartford Insurance Group Emergency Generator Exhaust Stack, 24. Hartford, CT, ROJAC Environmental Services, 1987.
- Compliance Test Report Southern New England Telephone Company Emergency Generator 25. Exhaust Stack, North Haven, CT, ROJAC Environmental Services, 1988.
- Compliance Test Report Pfizer, Inc. Two Emergency Generator Exhaust Stacks, Groton, CT, 26. ROJAC Environmental Services, 1987.
- L. M. Campbell, et al., Sourcebook: NO<sub>x</sub> Control Technology Data, Control Technology 27. Center, EPA-600/2-91-029, U. S. Environmental Protection Agency, Cincinnati, OH, July 1991.
- Catalysts For Air Pollution Control, Manufacturers Of Emission Controls Association 28. (MECA), Washington, DC, March 1992.

### Lehigh Cupertino kiln benzene data summary addendum to letter, August 20, 2014:

Below is a summary table showing recommended values for maximum hourly and annual average benzene emissions (lb/ton feed and lb/ton clinker), based on the 2009, 2011, and 2012 source test data summarized in Tables 1 and 2:



Note that the following changes were made to the 2012 data, based on observed outliers in data set:

- 1) 2012 RM-on high test—removed from 2012 RM-on average
- 2) 2012 RM-off high test-removed from 2012 RM-off average
- 3) 2012 RM-off production rate (feed and clinker rates)—values in average were replaced by using the same value as average for 2012 RM-on:

The RM-off production rates in the original test report are low, and do not seem realistic, given that the flow rates (corrected to 7%O2) are not also lower than for other tests.

K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\Review\benzene data summary addendum Aug 20 14.docx



Clinker rate is feed rate / 1.55, the feed/clinker factor derived from the 2012 source test. All calculated results within 10% of corresponding value in test report<br>Total flow is stack flow x 30.0, the flow factor derived from the 2012 source test.

K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\HRW to ATM deliv 8 4 14\Summary Tables v3.xlsx.xlsx

 $1 of 4$ 8/20/2014 9:50 AM



K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\HRW to ATM deliv 8 4 14\Summary Tables v3.xlsx.xlsx

2 of 4<br>8/20/2014 9:50 AM



RM On Run 1 and RM Off Run 2 are excluded because of anomalous benzene emissions (4 lb/hr)<br>RM Off feed and clinker rate are the average of RM On rates because measured rates are abnormally low for the flow rate (150 tph cl

 $3$  of  $4$ 8/20/2014 9:50 AM

K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\HRW to ATM deliv 8 4 14\Summary Tables v3.xlsx.xlsx

### Table 1 Calculation Parameters







K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\HRW to ATM deliv 8 4 14\Summary Tables v3.xlsx.xlsx





2012 RM On Run 1 and 2012 RM Off Run 2 are excluded because of anomalous benzene emissions (4 lb/hr)<br>2012 RM Off feed and clinker rate are the average of RM On rates because measured rates are abnormally low for the flow r

1 of 1<br>8/20/2014 9:50 AM

K:\11191.000.0\Stack Oct 12\Permit Apr 14\HRA comments\Benzene\HRW to ATM deliv 8 4 14\Summary Tables v3.xlsx.xlsx